Service Training



Self Study Program 826803 2.0 Liter TDI Common Rail BIN5 ULEV Engine



Volkswagen of America, Inc. Volkswagen Academy Printed in U.S.A. Printed 4/2008 Course Number 826803

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This Self-Study Program provides information regarding the design and function of new models.

This Self-Study Program is not a Repair Manual.

This information will not be updated. For maintenance and repair procedures, always refer to the latest electronic service information. Page intentionally left blank

A New Generation of Diesel Engines from Volkswagen

The 2.0 Liter TDI engine with common rail injection system is the first of a new generation of dynamic and efficient diesel engines from Volkswagen.

By combining the successful and proven 2.0 Liter TDI engine with common rail technology, Volkswagen is setting new standards in terms of such characteristic TDI attributes as dynamics, driving enjoyment, economy, and reliability. The superior qualities of the 2.0 Liter TDI engine with common rail injection system are oriented towards future challenges in acoustics, comfort, and exhaust gas after-treatment.

The lead taken on by Volkswagen in 1993 with the introduction of the first turbocharged direct injection (TDI) diesel engine in a passenger car continues with the 2.0 Liter TDI engine, confirming Volkswagen's role as a pioneer in diesel technology.

The engine offers the potential for future improvements in exhaust gas standards and the associated technologies.



Notes

Heritage

The 2.0 Liter TDI engine with common rail injection system is based on the 1.9 Liter TDI engine with the Unit Injector System (UIS) also known as the "pumpe düse." This predecessor engine is one of the most frequently built diesel engines in the world and has seen the broadest use within the Volkswagen Group, from passenger cars to transport vehicles. To accommodate the increasing demand for improvements in acoustics, fuel consumption, and exhaust gas emissions, a large number of engine components were redesigned. The conversion of the injection system to a common rail design is one of the major changes to this engine. Equipped with a special after-treatment system, this engine meets current emissions standards.





Technical Characteristics

- Common rail injection system with Piezo fuel injectors
- Diesel particulate filter with upstream oxidation catalyst
- Intake manifold with flap valve control
- Electric exhaust gas return valve
- Adjustable exhaust gas turbocharger with displacement feedback
- Low and high pressure Exhaust Gas Recirculation (EGR) system

Design	4-Cylinder In-Line Engine								
Displacement	120 in ³ (1968 cm ³)								
Bore	3.189 in. (81 mm)								
Stroke	3.760 in. (95.5 mm)								
Valves per Cylinder	4								
Compression Ratio	16.5:1								
Maximum Output	140 hp (103 kW) at 4000 rpm								
Maximum Torque	236 lb-ft (320 Nm) at 1750 rpm up to 2500 rpm								
Engine Management	Bosch EDC 17 (Common Rail Control Unit)								
Fuel	ULSD / ASTM D975-06b 2-D-S<15 (Ultra-Low Sulfur Diesel, under 15 ppm)								
Exhaust Gas Treatment	High and Low Pressure Exhaust Gas Return, Oxidation Catalytic Converter, Diesel Particulate Filter, NOx Storage Catalytic Converter								

2.0 Liter TDI Technical Data



2.0 Liter TDI Torque and Power



Crankshaft

The 2.0 Liter TDI common rail engine uses a forged crankshaft to accommodate high mechanical loads.

Instead of the customary eight counterweights, this crankshaft has only four. Using four counterweights reduces the load on the crankshaft bearings, as well as noise emissions that can occur due to the intrinsic motion and vibrations of the engine.

Pistons

The 2.0 Liter TDI common rail engine pistons have no valve pockets. This reduces the cylinder clearance and improves the swirl formation in the cylinder. Swirl is the circular flow about the vertical axis of the cylinder. Swirl has a significant influence on the mixture formation.

For cooling the piston ring zone, the piston has an annular cooling channel into which piston spray jets inject oil.

The piston bowl, where the injected fuel is circulated and mixed with air, is matched with the spray pattern of the injection jets and has a wider and flatter geometry than the piston in a pump-injection engine. This allows more homogeneous carburation and reduces soot formation.





Cylinder Head

The 2.0 Liter TDI common rail engine has a crossflow aluminum cylinder head with two intake and two exhaust valves per cylinder. The valves are arranged vertically upright.

The two overhead camshafts are linked by spur gears with an integrated backlash adjuster. They are driven by the crankshaft with a toothed belt and the exhaust camshaft timing gear. The valves are actuated by lowfriction roller cam followers with hydraulic valve lash adjusters.



Four-Valve Technology

Two intake and two exhaust valves per cylinder are arranged vertically suspended in the cylinder head. The vertically suspended and centrally situated fuel injector is arranged directly over the center of the piston bowl.

Shape, size, and arrangement of the intake and exhaust channels ensure a good degree of fill and a favorable charge cycle in the combustion chamber. The intake ports are designed as swirl and fill channels. The air flowing in through the fill channel produces the desired high level of charge motion.

The swirl channel ensures good filling of the combustion chamber, particularly at high engine speeds.



Intake Manifold with Flap Valves

Infinitely variable flap valves are located in the intake manifold.

Through the positioning of the flap valves, the swirl of the intake air is adjusted based on the engine speed and load.

The flap valves are moved by a pushrod connected to the Intake Flap Motor V157. This step motor is activated by the Engine Control Module (ECM) J623. The Intake Manifold Runner Position Sensor G336 is integrated in the Intake Flap Motor V157 and electronically regulates its movement. It also provides the Engine Control Module (ECM) J623 with feedback of the current position of the flap valves. Intake Manifold

Intake Flap Motor V157

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Design



Flap Valve Function

During idling and at low engine speeds, the flap valves are closed. This leads to high swirl formation, with results in good mixture formation.



During driving operation, the flap valves are adjusted continuously based on the load and engine speed. Thus for each operating range the optimum air movement is available.

Starting at an engine speed of approximately 3000 rpm, the flap valves are completely open. The increased throughput of air insures good filling of the combustion chamber.



At startup, during emergency operation, and at full load the flap valves are opened.



Camshaft Operation

The intake and exhaust camshafts are linked by means of spur gearing with an integrated backlash adjuster. The spur gear on the exhaust camshaft drives the spur gear on the intake camshaft.

Valve lash compensation ensures quiet camshaft operation.



Ramps

Ladder Frame

Design

The wider part of the spur gear (stationary spur gear) is a press-fit on the exhaust camshaft. There are ramps on the front face of the stationary spur gear.

The narrower part of the spur gear (the moving spur gear) can move in both radial and axial directions. There are recesses for the stationary spur gear ramps in the rear face of the moving spur gear.

How it Works

Both parts of the spur gear are pushed together in an axial direction by the force of a disk spring. At the same time, they are rotated by the ramp. Axial Direction Axial Direction Disk Spring

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The rotation leads to a gear displacement of the two spur gear parts and effects the lash adjustment between the intake and exhaust camshaft gears.



Cylinder Head Gasket

The cylinder head gasket is a four-layer design and has two special attributes that improve the sealing of the combustion chambers.

- Vertically profiled combustion chamber seals
- Rear flank support



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Vertically Profiled Combustion Chamber Seals

The sealing edge at the cylinder bore is referred to as the combustion chamber seal. It is vertically profiled, which means that the edge profile has varying heights around the perimeter of the combustion chamber. This special geometry provides for the uniform distribution of cylinder head gasket sealing forces around the combustion chambers. This prevents deformation at the cylinder bores and fluctuations in the sealing gap.

Rear Flank Support

The profile in the area of the two outer cylinders of the cylinder head gasket are referred to as "rear flank support." The rear flank support effects a uniform distribution of the gasket sealing forces in these areas. This reduces flexing of the cylinder head and deformation of the outer cylinders.



Toothed Belt Drive

The camshaft, the coolant pump, and the highpressure pump for the common rail injection system are driven by a toothed belt.



Accessory Drive

The generator and air conditioning compressor are driven by a ribbed V-belt. The profile surface of the ribbed V-belt has a fibrous coating. This improves the frictional properties of the belt, reducing unpleasant noise that can occur in wet and cold conditions.



Balance Shaft Module

The balance shaft module is installed below the crankshaft in the oil pan. The balance shaft module is driven by the crankshaft by a gear drive. The duocentric oil pump is integrated in the balance shaft module.

Design

The balance shaft module consists of a gray cast iron housing, two counter-rotating balance shafts, a helical-toothed gear drive, and an integrated duocentric oil pump. The rotation of the crankshaft is transferred to the intermediate gear on the outside of the housing. This drives the first balance shaft. From this balance shaft, the motion is then transferred inside the housing to the second balance shaft and to the duocentric oil pump. The gear drive is designed so that the balance shafts rotate at double the crankshaft speed.

The tooth backlash of the gear drive is adjusted with the help of a coating on the intermediate gear. This coating wears off during startup of the engine and results in excellent mating of the teeth on the two gears.



The intermediate gear must always be replaced if the intermediate gear or the drive gear of the first balance shaft have been loosened.

Please refer to the instructions in the Repair Manual.



Oil Circuit

A duocentric oil pump generates the oil pressure required for the engine. It is integrated into the balance shaft module and is driven by a balance shaft drive shaft.

The pressure relief valve is a safety valve. It prevents damage to engine components from excessive oil pressure, such as at high speeds and low ambient temperatures. The oil pressure control valve regulates the oil pressure in the engine. It opens as soon as the oil pressure reaches the maximum permissible value.

The bypass valve opens when the oil filter is clogged to safeguard the lubrication of the engine.



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Legend

- 1 Oil Pan
- 2 Oil Level and Temperature Transmitter
- 3 Oil Pump
- 4 Oil Pressure Relief Valve
- 5 Oil Return Block
- 6 Oil Cooler
- 7 Oil Filter
- 8 Bypass Valve

- 9 Oil Pressure Switch F1
- 10 Oil Pressure Control Valve
- 11 Crankshaft
- 12 Spraying Nozzles for Piston Cooling
- 13 Camshaft
- 14 Vacuum Pump
- 15 Turbocharger
- 16 Oil Return

Crankcase Ventilation

In combustion engines, pressure differentials between the combustion chamber and the crankcase generate air flow between piston rings and cylinder barrel, which are referred to as blow-by gases. These oily gases are returned to the intake area through the crankcase ventilation system to prevent pollution.

Effective oil separation keeps engine oil in the crankcase and prevents it from entering the intake manifold. This multistage system separates more oil than a single-stage system.

The oil separation is effected in three stages:

- Coarse separation
- Fine separation
- Damping section

The crankcase ventilation components, the oil filler inlet, and the pressure reservoir for the vacuum system of the engine are all integrated in the cylinder head cover.

Coarse Separation

The blow-by gases move from the crankshaft and camshaft chamber into a stabilizing section, which is integrated in the cylinder head cover. In this section, the larger oil droplets are separated onto the walls and collect on the floor. The oil can drip into the cylinder head through the openings in the stabilizing section.





Fine Separation

The fine separation takes place over a cyclone separator consisting of a total of four cyclones. Depending on the amount of the pressure differential between the intake manifold and the crankcase, two or four cyclones are activated by spring steel flutter valves.

Due to the geometry of the cyclones, the air is set into a rotating motion. The resulting centrifugal force slings the oil mist onto the separator wall. The oil droplets are deposited on the wall of the cyclone and are captured in a collector section.

When the engine is OFF, a flutter valve opens. This valve closes during engine operation due to the increased pressure in the cylinder head. The sole purpose of this valve is to let oil drain back into the engine sump when the engine is OFF.



Pressure Control Valve

The pressure control valve regulates the pressure for ventilation of the crankcase. It consists of a diaphragm and a pressure spring.

When blow-by gases are present, the pressure control valve limits the vacuum in the crankcase. Excessive vacuum in the crankcase could result in damage to the engine seals.

If the vacuum in the intake port is too small, the valve opens through the force of the pressure spring.

Pressure Control Valve Opened



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If the vacuum in the intake port is too large, the pressure control valve closes.

Pressure Control Valve Closes



Damping Section

To prevent disruptive swirl upon introduction of the gases in the intake manifold, a damping section connects to the cyclone oil separator. In this section the motion energy of the gases from the cyclone is reduced, and a residual quantity of oil is again separated out.



Coolant Circuit

In the coolant circuit, the coolant is circulated by a mechanical coolant pump. It is driven by the toothed belt. The circuit is controlled by an expansion-element thermostat, the coolant control unit.



Legend

- 1 Radiator
- 2 Coolant Control Unit (Expansion Element Thermostat)
- 3 Coolant Pump
- 4 Transmission Cooler (if applicable)
- 5 Oil Cooler
- 6 Cooler for Exhaust Gas Return
- 7 Heat Exchanger For Heater
- 8 Equalizing Reservoir
- 9 -- Engine Block Heater (optional)



The engine block heater is not planned to be available until early in 2009 and will be a dealer-installed item.

High and Low Pressure Exhaust Gas Recirculation (EGR) System

The most effective measure to reduce nitrogen oxides (NOx) with an internal combustion engine is by introducing very high exhaust gas recirculation rates into the combustion chamber. An additional advantage is to introduce these very high exhaust gases at very low temperatures. The current cooled EGR systems that exist in many applications today had to be modified. To meet BIN 5 emission standards, the entire operating characteristics of the engine up to full-load required EGR operation.



The air mass regulation of the High-Pressure EGR is regulated by the EGR Vacuum Regulator Solenoid Valve N18 and servo and by the turbocharger vane direction. The short path of the High-Pressure EGR is used in order to reach the desired EGR rate while driving at lower engine speeds and loads.

The combined EGR operation is continuously adjusted depending on engine operating conditions and revolutions-per-minute (RPM). Thus, no-load engine operation results in high amounts of High Pressure EGR application. With rising engine load and engine RPM, the recirculation of exhaust gases is shifted to the Low Pressure EGR system to increase the recirculation rate. This happens in order to obtain optimal NOx reduction at middle and high engine loads. Particularly in the high engine loads, the cooled Low Pressure EGR is a very large advantage over the High Pressure EGR system.



Uncooled EGR001

The Fuel System

Schematic Overview

1 - Transfer Fuel Pump (FP) G6

Feeds fuel continuously in the presupply area (from the fuel tank).

2 - Fuel Filter with Preheating Valve

The preheating valve prevents the filter from becoming clogged due to crystallization of paraffin in low ambient temperatures.

3 – Auxiliary Fuel Pump V393

Feeds fuel from the presupply area to the fuel pump.

4 - Filter Screen

Protects the high-pressure pump from dirt particles.

5 - Fuel Temperature Sensor G81

Determines the current fuel temperature.

6 – High-Pressure Pump

Generates the high fuel pressure needed for injection.

7 - Fuel Metering Valve N290

Regulates the quantity of fuel to be compressed based on demand.





8 – Fuel Pressure Regulator Valve N276

Adjusts the fuel pressure in the high-pressure area.

9 - High-Pressure Accumulator (Rail)

For all cylinders, stores the fuel needed for injection under high pressure.

10 - Fuel Pressure Sensor G247

Determines the current fuel pressure in the highpressure area.

11 – Pressure Retention Valve

Retains the return pressure of the fuel injectors at approximately 145 psi (10 bar). This pressure is needed for the function of the fuel injectors.

12 – Cylinder 1 through 4 Fuel Injectors N30, N31, N32, N33



Common Rail Injection System

The common rail injection system is a high-pressure accumulator injection system for diesel engines. The term "common rail" refers to the shared fuel high-pressure accumulator for all fuel injectors in a cylinder bank.

In this type of injection system, pressure generation and fuel injection are performed separately. A separate high-pressure pump generates the high fuel pressure required for injection. This fuel pressure is stored in a high-pressure accumulator (rail) and supplied to the fuel injectors over short injection lines.

The common rail injection system is controlled by the Bosch EDC 17 engine management system.

High-Pressure Accumulator (Rail)



High-Pressure Pump

The characteristics of this injection system are:

- The injection pressure is selectable and can be adapted to the operating conditions of the engine.
- A high injection pressure up to a maximum of 26,107 psi (1800 bar) enables good mixture formation.
- A flexible course of injection with multiple pre- and post-injections.

The common rail injection system can adapt the injection pressure and the timing of the injection to the operating conditions of the engine.

This system is very well suited to fulfill the constantly increasing requirements for an injection system to provide greater fuel economy, lower emissions, and quiet operation.



Fuel Injectors N30, N31, N32, N33

In the common rail system of the 2.0 Liter TDI engine, piezo-controlled Fuel Injectors N30, N31, N32, and N33 are used.

The fuel injectors are controlled over a piezo actuator. The switching speed of a piezo actuator is approximately four times faster than a solenoid valve.

Compared to solenoid actuated fuel injectors, piezo technology also involves approximately 75% less moving mass at the nozzle pin.

This results in the following advantages:

- Very short switching times
- Multiple injections possible per work cycle
- Precise metering of injection quantities

Course of Injection

Due to the very short switching times of the piezocontrolled fuel injectors, it is possible to control the injection phases and quantities flexibly and precisely. This enables the course of injection to be adapted



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to the operating conditions of the engine. Up to five partial injections can be performed per course of injection.



Auxiliary Fuel Pump V393

The Auxiliary Fuel Pump V393 is a roller-cell pump. It is located in the engine compartment and has the task of feeding fuel from the fuel tank to the high-pressure pump. The Auxiliary Fuel Pump V393 is actuated by the Engine Control Module (ECM) J623 through a relay and increases the fuel pressure presupplied by the Transfer Fuel Pump (FP) G6 in the fuel tank to approximately 73 psi (5 bar).

Effects of Failure

If the Auxiliary Fuel Pump V393 fails, the engine runs at first with reduced power. An engine startup is not possible.



Auxiliary Fuel Pump V393

Filter Screen

To protect the high-pressure pump from dirt particles, a filter screen is installed before the high-pressure pump in the fuel inlet.



High-Pressure Pump

The high-pressure pump is a single-piston pump. It is driven via the toothed belt by the crankshaft at engine speed.

The high-pressure pump has the job of generating the high fuel pressure of up to 26,107 psi (1800 bar) needed for injection. Pressure is generated by the rotation of two cams offset by 180 degrees on the pump drive shaft. The injection is always in the operating cycle of the respective cylinder. This keeps the pump drive evenly loaded and pressure fluctuations in the high-pressure area are minimized.

Design of the High-Pressure Pump




When setting the control times of the engine, the position of the high-pressure pump drive shaft must be set. Please refer to the instructions in the Repair Manual.

Design of the High-Pressure Pump - Schematic



Engine Mechanics

High-Pressure Area

The high-pressure pump is supplied with adequate fuel by the Auxiliary Fuel Pump V393 in each operating range of the engine.

The fuel enters the high-pressure area of the engine through the Fuel Metering Valve N290.

The pump piston is moved upward and downward by the cams on the pump drive shaft.



Intake Stroke

The downward motion of the pump piston increases the volume the compression space.

This results in a pressure differential between the fuel in the high-pressure pump and the compression space.

The intake valve opens and fuel flows into the compression space.



Delivery Stroke

With the beginning of the upward motion of the pump piston, the pressure in the compression space increases and the intake valve closes.

As soon as the fuel pressure in the compression space exceeds the pressure in the high-pressure area, the exhaust valve (check valve) opens and fuel enters the high-pressure accumulator (rail).



Fuel Metering Valve N290

Fuel Metering Valve N290 is integrated in the highpressure pump. It ensures demand-based control of the fuel pressure in the high-pressure area. The Fuel Metering Valve N290 controls the fuel quantity that is needed for high-pressure generation. This represents an advantage, in that the high-pressure pump must generate only the pressure needed for the momentary operating situation. The power consumption of the high-pressure pump is reduced and unnecessary warming up of the fuel is avoided.

Function

The non-energized state the Fuel Metering Valve N290 is open. To reduce the feed quantity to the

compression space, the valve is actuated by the Engine Control Module (ECM) J623 with a pulsewidth modulated (PWM) signal.

Through the PWM signal the Fuel Metering Valve N290 is closed cyclically. Depending on the duty cycle, the position of the locking piston changes as does the amount of fuel into the compression space of the high-pressure pump.

Effects of Failure

Engine power is reduced. Engine management operates in emergency mode.



Low-Pressure Area

Overflow Valve

The fuel pressure in the low-pressure area of the high-pressure pump is controlled by the overflow valve.

Function

The Auxiliary Fuel Pump V393 delivers fuel from the fuel tank with a pressure of approximately 73 psi (5 bar) into the high-pressure pump. Thus the fuel supply to the high-pressure pump is ensured in all operating conditions.

The overflow valve regulates the fuel pressure in the high-pressure pump to approximately 62 psi (4.3 bar).

The fuel delivered by the Auxiliary Fuel Pump V393 acts in opposition to the piston and the piston spring of the overflow valve. With a fuel pressure over 62 psi (4.3 bar), the overflow valve opens and clears the way to the fuel return. The excess fuel flows through the fuel return into the fuel tank.



Control of the Fuel High Pressure

In the common rail injection system, the fuel high pressure is controlled by a so-called two-controller concept.

Depending on the operating conditions, the high fuel pressure is regulated either by the Fuel Pressure Regulator Valve N276 or the Fuel Metering Valve N290. The valves are actuated by the Engine Control Module (ECM) J623 with a pulse-width modulated (PWM) signal.

Control by the Fuel Pressure Regulator Valve N276

At engine start and for preheating of the fuel, the high fuel pressure is controlled by the Fuel Pressure Regulator Valve N276. To heat up the fuel guickly, the high-pressure pump delivers and compresses more fuel than is needed. The excess fuel is discharged by the Fuel Pressure Regulator Valve N276 into the fuel return.

Control by the Fuel Metering Valve N290

With large injection quantities and high rail pressures, the high fuel pressure is controlled by the Fuel Metering Valve N290. This effects a demand-based regulation of the fuel high pressure. The power consumption of the high-pressure pump is reduced and unnecessary heating up of the fuel is avoided.

Control by Both Valves

In idling, in trailing throttle condition, and with small injection quantities, the fuel pressure is controlled by both valves simultaneously. This enables precise control, which improves idling quality and the transition into trailing throttle condition.



Two-Controller Concept



Control of Fuel High Pressure by Fuel Pressure Regulator Valve N276

Control of Fuel High Pressure by Fuel Metering Valve N290

Control by Both Valves

Engine Mechanics

Fuel Pressure Regulator Valve N276

The Fuel Pressure Regulator Valve N276 is located on the high-pressure accumulator (rail).

Opening and closing of the Fuel Pressure Regulator Valve N276 adjusts the pressure of the fuel in the high-pressure area.

This is actuated by the Engine Control Module (ECM) J623 with a pulse-width modulated signal.



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Design

Engine Mechanics

How it Works

In contrast to conventional control valves in common rail injection systems, the Fuel Pressure Regulator Valve N276 is open in the non-energized state.

Fuel Pressure Regulator Valve N276 in Rest Position (Engine "Off")

If the Fuel Pressure Regulator Valve N276 is not activated, the pressure regulator valve is opened by the valve springs. The high-pressure area is connected to the fuel return.

This ensures volume compensation between the high-pressure and low-pressure areas. Fuel vapor lock, which can occur during the cool-down with engine standstill in the high-pressure accumulator (rail), is avoided and the startup properties of the engine are improved.

Fuel Pressure Regulator Valve N276 Activated (Engine "On")

To set an operating pressure of 3,336 to 26,107 psi (230 to 1800 bar) in the high-pressure accumulator, the Fuel Pressure Regulator Valve N276 is actuated by the Engine Control Module (ECM) J623 with a pulse-width modulated (PWM) signal. Upon actuation a magnetic field is generated in the solenoid coil. The valve anchor is tightened and presses the valve needle into its seat. A magnetic force opposes the fuel pressure in the high-pressure accumulator. Depending on the duty cycle of the actuation, the flow cross-section to the return line and the exhaust quantity is changed. This also allows fluctuations in the pressure in the high-pressure accumulator to be compensated.

Effects of Failure

If the Fuel Pressure Regulator Valve N276 fails, the engine cannot run because adequate high fuel pressure cannot be developed for injection.



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System Overview

Sensors

Engine Speed (RPM) Sensor G28 Camshaft Position (CMP) Sensor G40 Throttle Position (TP) Sensor G79 / Accelerator Pedal Position Sensor 2 G185 Mass Air Flow (MAF) Sensor G70 Engine Coolant Temperature (ECT) Sensor G62 Charge Air Pressure Sensor G31 Intake Air Temperature (IAT) Sensor G42 Manifold Absolute Pressure (MAP) Sensor G71 Fuel Temperature Sensor G81 Fuel Pressure Sensor G247 EGR Potentiometer G212 Heated Oxygen Sensor (HO2S) G39 Exhaust Pressure Sensor 1 G450 Low Pressure Exhaust Gas Recirculation (EGR) Pressure Sensor Exhaust Gas Temperature (EGT) Sensor 1 G235 Exhaust Gas Temperature (EGT) Sensor 2 G448 Exhaust Gas Temperature (EGT) Sensor 3 G495 Exhaust Gas Temperature (EGT) Sensor 4 G648 Exhaust Gas Recirculation (EGR) Temperature Sensor G98 Engine Coolant Temperature (ECT) Sensor (on radiator) G83 Oxygen Sensor (O2S) Behind Three Way Catalytic Converter (TWC) G130 Brake Light Switch F Clutch Position Sensor G476 Charge Pressure Actuator Position Sensor G581 Intake Manifold Runner Position Sensor G336 Cylinder Pressure Sensors G620 - G623 Throttle Position (TP) Sensor G69



Engine Management System



Actuators

Fuel Pump (FP) Relay J17 Transfer Fuel Pump (FP) G6

Auxiliary Fuel Pump Relay J832 Auxiliary Fuel Pump V393

Cylinder 1 Fuel Injector N30 Cylinder 2 Fuel Injector N31 Cylinder 3 Fuel Injector N32 Cylinder 4 Fuel Injector N33

Fuel Metering Valve N290

Fuel Pressure Regulator Valve N276

Wastegate Bypass Regulator Valve N75 (uses variable turbine geometry)

Intake Flap Motor V157

Throttle Valve Control Module J338

Exhaust Flap Control Module J883 With Position Sensor

EGR Vacuum Regulator Solenoid Valve N18

Exhaust Gas Recirculation (EGR) Cooler Switch-Over Valve N345

EGR Valve 2 N213

Engine Coolant (EC) Circulation Pump 2 V178

Oxygen Sensor (O2S) Heater Z19

Oxygen Sensor (O2S) Heater Z28

Automatic Glow Time Control Module J179 Glow Plug 1 Q10 Glow Plug 2 Q11 Glow Plug 3 Q12 Glow Plug 4 Q13

Engine Management System

Electronic Diesel Control (EDC) Engine Management

The engine management system of the 2.0 Liter TDI engine with common rail injection system is the electronic diesel control EDC17 from Bosch.

The EDC17 engine management system is the successor of EDC16. EDC17 has greater processing capability and a larger storage capacity than EDC16.

It also offers the option of integrating control functions for future technologies.

Control Devices in the CAN Data Bus

The schematic below shows the integration of the Engine Control Module J623 into the CAN data bus structure of the vehicle. Information is transmitted between control devices over the CAN data bus.





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Legend

- J104 ABS Control Module
- J217 Transmission Control Module (TCM)
- J234 Airbag Control Module
- J285 Instrument Cluster Control Module
- J519 Vehicle Electrical System Control Module
- J527 Steering Column Electronic Systems Control Module
- J533 Data Bus On Board Diagnostic Interface
- J623 Engine Control Module (ECM)

Exhaust Gas Turbocharger

The boost pressure in the 2.0 Liter TDI engine is generated by an adjustable turbocharger. It has adjustable guide vanes that can be used to influence the flow of exhaust gas onto the turbine wheel. The advantage is that optimum boost pressure and good combustion are achieved over the entire engine speed range. The adjustable guide vanes ensure high torque and good starting behavior in the lower speed range, as well as low fuel consumption and low exhaust gas emissions in the upper speed range. A linkage controlled by vacuum is used to adjust the guide vanes.



Flow Damper

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Flow Damper

A flow damper is located behind the outlet of the turbocharger in the charge air section. It has the task of reducing disagreeable noise from the turbocharger, such as whistling.

Design and Function

During full-load acceleration the turbocharger must build up boost pressure very quickly. The turbine and compressor wheel are accelerated quickly and the turbocharger approaches its pump limit. This can lead to burbling in the air flow, which causes disturbing noise that radiates into the charge air section.

The charge air causes the air in the resonance sections of the flow damper to vibrate. The vibration has approximately the same frequency as the noise in the charge air. Disturbing noise is minimized by superimposition of the charge air sound waves with the vibration of the air in the resonance sections of



Boost Pressure Control

The boost pressure control manages the volume of air that is compressed by the turbocharger.



Legend

- 1 Vacuum System
- 2 Engine Control Module (ECM) J623
- 3 Intake Air
- 4 Charge Air Cooler
- 5 Vacuum Regulator Valve
- 6 Turbocharger Compressor
- 7 Actuator / Guide Vane Position Sensor
- 8 Exhaust Gas Turbine with Guide Vane Adjuster
- 9 Diesel Particulate Filter / Oxidation Catalyst
- 10 Charge Air Pressure Sensor G31 and Intake Air Temperature (IAT) Sensor G42
- 11 Low Pressure EGR Valve

Wastegate Bypass Regulator Valve N75

The Wastegate Bypass Regulator Valve N75 is an electro-pneumatic valve. This valve is used to control the vacuum needed to adjust the guide vanes over the vacuum cell.

Effects of Failure

If the Wastegate Bypass Regulator Valve N75 fails the vacuum cell is not supplied with vacuum. A spring in the vacuum cell pushes the linkage of the adjusting mechanism so that the guide vanes of the turbocharger are brought into a steep approach angle (emergency mode position). With lower engine speed and thus lower exhaust gas pressure, only a low boost pressure is available. The engine has less power, and an active regeneration of the particulate filter is not possible.



Wastegate Bypass Regulator Valve N75

Cylinder Pressure Sensors

A cylinder pressure sensor is integrated into each of the Glow Plugs. The glow element is attached to an extension, which can apply pressure to a diaphragm. The diaphragm has strain gauges that change resistance by deformation. The integrated electronics calculate tension, which is proportional to the combustion chamber pressure.

Signal Use

The pressure sensor collects cylinder burn-data such as the burn moment and the situation of the burn in relation to the crankshaft. This can result in an increase or decrease of the injection amount, as the pressure is indirectly related to the injection amount. Correcting the injection using pressure sensor information balances the injection for all cylinders. In addition, this correction applies to manufacturing tolerances and engine aging.

As a direct result of the pressure sensors, emission tolerances are clearly reduced over the life span of the engine.

The regulation of the burn is accomplished by shifting the start of injection. Thus, the burn stabilizes during times of very large exhaust recirculation rates and misfires and other running issues can be avoided. Also, the pressures can help to balance the time delays caused by bad fuel (low Cetane).

Effect of Failure

If one of the pressure sensors fails, a substitute value will be used from the other pressure sensors.



Glow Plug / Pressure Sensor Connector



Charge Air Pressure Sensor G31 and Intake Air Temperature (IAT) Sensor G42

Charge Air Pressure Sensor G31 and Intake Air Temperature (IAT) Sensor G42 are integrated in one component and are located in the intake manifold.

Charge Air Pressure Sensor G31

Signal Use

The air pressure in the intake manifold is determined from the Charge Air Pressure Sensor G31 signal. Engine Control Module (ECM) J623 needs the signal to control the boost pressure.

Effect of Failure

If the Charge Air Pressure Sensor G31 signal fails, there is no substitute function. The boost pressure control is disengaged and the engine power decreases significantly. The particulate filter cannot be actively regenerated.

Intake Air Temperature (IAT) Sensor G42

Engine Control Module (ECM) J623 uses the signal of Intake Air Temperature (IAT) Sensor G42 to control the boost pressure. Because the temperature influences the density of the charge air, the signal is used by Engine Control Module (ECM) J623 as a correction value.





Charge Pressure Actuator Position Sensor G581

Charge Pressure Actuator Position Sensor G581 is integrated in the vacuum cell of the turbocharger. It is a displacement sensor that enables Engine Control Module (ECM) J623 to determine the position of the guide vanes in the turbocharger.

Signal Use

The signal of Charge Pressure Actuator Position Sensor G581 delivers the position of the guide vanes of the turbocharger to Engine Control Module (ECM) J623. Together with the signal of Charge Air Pressure Sensor G31, this allows conclusions about the state of boost pressure control.

Effects of Failure

If Charge Pressure Actuator Position Sensor G581 fails, the signal from Charge Air Pressure Sensor G31 and the engine speed are used to determine the position of the guide vanes. Malfunction Indicator Lamp (MIL) K83 is actuated.



EGR Vacuum Regulator Solenoid Valve N18

The EGR Vacuum Regulator Solenoid Valve N18 is an electric motor controlled valve plate. It is actuated by the Engine Control Module (ECM) J623 and can be infinitely adjusted by the electric motor. The angle of the valve plate controls the quantity of returned exhaust gas.

Effect of Failure

If EGR Vacuum Regulator Solenoid Valve N18 fails, the valve plate is closed by a valve spring. No exhaust gas can be returned.

EGR Potentiometer G212

EGR Potentiometer G212 captures the position of the valve plate in the exhaust gas return valve.

Signal Use

Based on the signal, Engine Control Module (ECM) J623 recognizes the position of the valve plate. This enables control of returned exhaust gas volume and thus the nitrogen oxide content in the exhaust gas.

Effect of Failure

If EGR Potentiometer G212 fails, the exhaust gas return is deactivated. The EGR Vacuum Regulator Solenoid Valve N18 drive is switched to the nonenergized state and the valve plate is closed by a valve spring.



Exhaust Gas Return Valve with EGR Potentiometer G212 and EGR Vacuum Regulator Solenoid Valve N18

Engine Management System

Low Pressure Exhaust Gas Recirculation (EGR) Valve with EGR Potentiometer and EGR Valve N345

The exhaust gas return cooler is a switchable cooler that allows the engine and diesel particulate filter to reach their operating temperatures more quickly. The exhaust gas cooler is activated when the coolant temperatures reach 99°F (37°C)

The Exhaust Gas Recirculation (EGR) Cooler Switch-Over Valve N345 is an electrically-controlled valve plate . It is actuated by the Engine Control Module (ECM) J623 and can be infinitely adjusted by the electric motor. The position of the valve plate controls the quantity of the returned exhaust gas.

Effect of Failure

If the Exhaust Gas Recirculation (EGR) Cooler Switch-Over Valve N345 fails, the valve plate is closed by a spring. No exhaust gas can be returned.



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To the Intake Side of the Turbocharger

Low Pressure Exhaust Gas Recirculation Valve with EGR Potentiometer and EGR Valve N345

EGR Potentiometer

The EGR Potentiometer captures the position of the valve plate in the Low Pressure exhaust gas recirculation valve.

Signal Use

Based on the signal, the Engine Control Module (ECM) J623 recognizes the position of the valve plate. This enables control of the returned exhaust gas volume and thus the nitrogen oxide content in the exhaust gas.

Effect of Failure

If the EGR potentiometer fails, the Low Pressure recirculation is deactivated. The Low Pressure EGR Vacuum Regulator Solenoid Valve N345 drive is switched to the non-energized state and the valve plate is closed by the valve spring

Throttle Valve Control Module J338

In the direction of flow, Throttle Valve Control Module J338 is mounted before EGR Vacuum Regulator Solenoid Valve N18.

There is an electric motor in Throttle Valve Control Module J338 that moves the throttle with a gear. Adjustment of the throttle is infinite and can be adapted to the respective load and speed of the engine.

The Throttle Valve Control Module J338 has the following tasks:

In certain operating situations, a differential between intake manifold pressure and exhaust gas pressure is generated through the throttle. This pressure differential facilitates exhaust gas return.

In the regeneration mode of the diesel particulate filter, the intake air volume is regulated with the throttle. When the Throttle Valve Control Module J338 motor is switched off, the throttle is closed. Less air is taken in and compressed, and the engine shuts down smoothly.

Effect of Failure

If Throttle Valve Control Module J338 fails, correct regulation of the exhaust gas return rate is not possible. An active regeneration of the diesel particulate filter does not take place.



Throttle Valve Control Module J338 with Throttle Position (TP) Sensor G69

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Throttle Position (TP) Sensor G69

Throttle Position (TP) Sensor G69 is integrated in the throttle drive. The sensor element captures the position of the throttle.

Signal Use

Based on the signal, Engine Control Module (ECM) J623 recognizes the position of the throttle. This information is needed for control of exhaust gas return and particulate filter regeneration.

Effect of Failure

If Throttle Position (TP) Sensor G69 fails, the exhaust gas return is deactivated and active regeneration of the diesel particulate filter does not take place.

Exhaust Throttle Valve

The exhaust throttle valve is a new component. In the direction of exhaust flow, the Exhaust Throttle Valve is located behind the NOx storage catalytic converter.

There is an electric motor inside of the Exhaust Throttle Valve that moves the throttle plate with a gear. Adjustment of the throttle plate is infinite and can be adapted to respective load and speed of the engine.

The Exhaust Throttle Valve has the following tasks:

- In certain operating conditions, a differential pressure is generated between the NOx storage catalyst and the turbocharger.
- This increase in pressure helps with Low Pressure EGR return.



Effect of Failure

If the Throttle Valve Control Module fails, the correct regulation of exhaust gas recirculation rate is not possible. Regeneration of the NOx storage catalyst does not take place.

Throttle Valve with Throttle Position (TP) Sensor

The throttle Position Sensor is integrated into the throttle valve drive. The purpose of this sensor is to capture the position of the throttle valve.

Signal Use

Based on the signal, Engine Control Module (ECM) J623 recognizes the position of the throttle. This information is needed for control of exhaust gas recirculation

Effect of Failure

If Throttle Position (TP) Sensor G69 fails, the exhaust gas recirculation is deactivated.

Exhaust System

The exhaust system of the 2.0L Common-Rail is very different from previous engines. The exhaust system consists of the following main components:

- Oxidation Catalytic Converter
- Particulate Filter
- Nitrogen Oxide Catalytic Converter
- H₂S Catalytic Converter

Oxidation Catalytic Converter

Diesel Particulate Filter System

In addition to internal engine measures in the 2.0 Liter TDI engine with common rail injection system, soot particle emissions are further reduced through a diesel particulate filter.

The diesel particulate filter is found together with the oxidation catalyst in a housing. It is located close to the engine so that it will reach operating temperature quickly.

Engine Management System

Diesel Particulate Filter and NOx Storage Catalytic Converter System Overview

Legend

- 1 Mass Air Flow (MAF) Sensor G70
- 2 Exhaust Gas Temperature (EGT) Sensor 1 G235
- 3 Turbocharger
- 4 Heated Oxygen Sensor (HO2S) G39
- 5 Oxidation Catalyst
- 6 Particulate Filter
- 7 Exhaust Gas Temperature (EGT) Sensor 3 G495
- 8 Exhaust Pressure Sensor 1 G450
- 9 Exhaust Gas Temperature (EGT) Sensor 4 G648
- 10 Cylinder Pressure Sensors 1,2,3,4
- 11 Low Pressure EGR Temperature Sensor
- 12 Low Pressure EGR Potentiometer
- 13 Exhaust Gas Temperature (EGT) Sensor 2
- 14 Heated Oxygen Sensor (H02S) G130

Additional Components not Pictured:

- Low Pressure EGR Differential Pressure Sensor
- Wastegate Bypass Regulator Valve N75

Diesel Particulate Filter System Design

The diesel particulate filter and the oxidation catalyst are installed separately in a shared housing. The oxidation catalyst is located before the particulate filter in the direction of flow.

This design with the oxidation catalyst upstream offers the following advantages in connection with the common rail injection system:

- Because of the upstream placement of the oxidation catalyst, the temperature of the exhaust gas is increased before it enters the diesel particulate filter. As a result, the operating temperature of the diesel particulate filter is reached quickly.
- In trailing throttle condition, over cooling of the diesel particulate filter by the cold intake air is prevented. In this case, the oxidation catalyst acts as a heat exchanger, from which the warmth is routed through the exhaust gas flow to the particulate filter.
- In the regeneration operation, the temperature of the exhaust gas is accurately controlled. The Exhaust Gas Temperature (EGT) Sensor 3 G495 determines the temperature of the exhaust gas directly before the particulate filter. As a result, the fuel quantity of the post-injection is precisely calculated to increase the exhaust gas temperature in the regeneration operation.

Oxidation Catalyst

The carrier material of the oxidation catalyst is metal, so the light-off temperature is reached quickly. This metal body has an aluminum oxide carrier coating, onto which platinum and palladium are vapordeposited as catalyst for the hydrocarbons (HC) and the carbon monoxide (CO).

Function

The oxidation catalyst converts a large portion of the hydrocarbons (HC) and the carbon monoxide (CO) into water vapor and carbon dioxide.

Diesel Particulate Filter

The diesel particulate filter consists of a honeycombshaped ceramic body made of aluminum titanide. The ceramic body is partitioned into a large number of small channels, which are alternately open and closed at the ends. This results in inlet and outlet channels that are separated by filter walls.

The filter walls are porous and coated with a carrier coating of aluminum oxide. Vapor-deposited onto this carrier layer is the precious metal platinum, which acts as catalyst.

Function

As the soot-containing exhaust gas flows through the porous filter walls of the inlet channels, the soot particles are captured in the inlet channels.

Regeneration

The particulate filter must be regenerated regularly so that it does not become clogged with soot particles and its function impaired. During regeneration, the soot particles collected in the particulate filter are burned off (oxidized).

The regeneration of the particulate filter is performed in the following stages:

- Warm-up phase
- Passive regeneration
- Active regeneration
- Customer-initiated regeneration drive
- Service regeneration

Warm-Up Phase

To heat up a cold oxidation catalyst and particulate filter as quickly as possible and thus bring them to operating temperature, the engine management system introduces a post-injection after the main injection.

This fuel combusts in the cylinder and increases the combustion temperature. Through the air flow in the exhaust gas tract, the resulting heat reaches the oxidation catalyst and the particulate filter and heats them.

The warm-up phase is complete when the operating temperature of the oxidation catalyst and the particulate filter has been reached for a specific period of time.

Passive Regeneration

During passive regeneration the soot particles are continuously burned without the intervention of Engine Control Module (ECM) J623.

This occurs primarily at higher engine load, such as in highway driving, when exhaust gas temperatures range from 662°F to 932°F (350°C to 500°C).

At these temperatures the soot particles are converted into carbon dioxide through a combustion reaction with nitrogen dioxide.

Active Regeneration

In a large portion of the operating range the exhaust gas temperatures are too low for a passive regeneration. Because soot particles can no longer be eliminated passively, soot accumulates in the filter. As soon as a specific soot load has been reached in the filter, the Engine Control Module (ECM) J623 initiates an active regeneration. The soot particles are burned off at an exhaust gas temperature of 1022°F to 1202°F (550°C to 650°C).

Active Regeneration Function

The soot load of the particulate filter is calculated by two pre-programmed load models in the Engine Control Module (ECM) J623.

One of the load models is determined from the driving profile of the user and the signals from the exhaust gas temperature sensors and Heated Oxygen Sensor (HO2S) G39.

Another soot load model is the flow resistance of the particulate filter. It is calculated from the signals of Exhaust Pressure Sensor 1 G450, Exhaust Gas Temperature (EGT) Sensor 3 G495, and Mass Air Flow (MAF) Sensor G70.

Engine Control Module (ECM) J623 has several ways to control the increase of exhaust gas temperatures during active regeneration:

- The intake air supply is regulated by Throttle Valve Control Module J338.
- The exhaust gas return is deactivated to increase the combustion temperature and the oxygen content in the combustion chamber.
- Shortly after a delayed "late" main injection, the first post-injection is initiated to increase the combustion temperature.
- Late after the main injection an additional postinjection is initiated. This fuel does not combust in the cylinder, but instead vaporizes in the combustion chamber.
- The unburned hydrocarbons of this fuel vapor are oxidized in the oxidation catalyst. This ensures an increase in the exhaust gas temperature to approximately 1202°F (650°C) as it reaches the particulate filter.
- To calculate the injection quantity for the late post-injection, Engine Control Module (ECM) J623 uses the signal of Exhaust Gas Temperature (EGT) Sensor 3 G495 located before the particulate filter.
- The boost pressure is adjusted so that the torque during the regeneration operation does not change noticeably for the driver.

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Customer-Initiated Regeneration Drive

An exhaust gas temperature high enough for particulate filter regeneration is not reached when the vehicle is only driven for short-distances. If the load condition of the diesel particulate filter reaches a threshold value, Diesel Particle Filter Indicator Lamp K231 in the instrument panel will light up.

This signal prompts the driver to perform a regeneration drive. The vehicle must be driven for a short period of time at increased speed to ensure that an adequately high exhaust gas temperature is reached. The operating conditions must remain constant over the period for a successful regeneration.

Details of the driving behavior required when the Diesel Particle Filter Indicator Lamp K231 comes on can be found in the Owner's Manual.

Service Regeneration

If the regeneration drive is not successfully completed and the load condition of the diesel particulate filter has reached 1.41 ounces (40 grams), Diesel Particle Filter Indicator Lamp K231 and Glow Plug Indicator Lamp K29 will light up simultaneously.

The text "Check Engine – Service Shop" will appear in the instrument panel display.

This prompts the driver to visit the nearest service shop. In this case, the Engine Control Module (ECM) J623 blocks active regeneration of the diesel particulate filter to prevent damage to the filter and the particulate filter can only be regenerated by service regeneration with the VAS 5051.

When the load condition reaches 1.59 ounces (45 grams), service regeneration is no longer possible. Because the danger of destroying the filter is too great with this load, the filter must be replaced.

Regeneration of the 2.0 Liter TDI Particulate Filter

Distance Regeneration

"Distance regeneration" is a distance-dependent regeneration of the particulate filter. The Engine Control Module (ECM) J623 initiates an active regeneration automatically if during the last 466 to 621 miles (750 to 1000 km) of travel no successful regeneration has taken place, regardless of the load condition in the diesel particulate filter.

Distance regeneration serves as additional safeguard to minimize the load condition of the diesel particulate filter.

Nitrogen Oxide Catalytic Converter

To attain the BIN5/LEV2 emission level, an efficient system for exhaust gas after-treatment is required. The NOx storage catalyst is used to supplement the particulate filter system.

By placing the NOx storage catalytic converter away from the engine in the vehicle underbody, the thermal aging is considerably reduced. This also takes advantage that the CO and HC that have already been oxidized by the particulate filter. This allows an optimum NOx conversion in the NOx catalytic converter.

Oxidation Catalytic Converter

The exhaust system has two lambda sensors. The lambda sensor upstream of the oxidation catalytic converter regulates the air-reduced operating modes for the NOx catalytic converter. It is also used for the initial value for the air model stored in the engine control unit. This air model help to determine the model-based NOx and soot emissions of the engine. The second lambda sensor, which is placed downstream of the NOx catalytic converter, detects an excess of reduction medium in the regeneration phase. This is used to determine loading and the aging condition of the NOx catalytic converter.

The three temperature sensors integrated into the exhaust system enable the OBD functions for the catalytic components and are used as initial values in the regulation of the regeneration operating modes ad the exhaust temperature model.

Additional Engine Operating Modes for Exhaust After-Treatment

DeNox Mode

The enhancement of the exhaust after-treatment system with a NOx storage catalytic converter requires the introduction of new regeneration modes to ensure NOx conversion throughout the storage unit's service life.

Unlike particulate filter regeneration, a substoichiometric exhaust gas composition is necessary for the regeneration of the NOx storage catalytic converter. In sub-stoichiometric operation, the nitrogen oxides stored during the lean operation are reduced by the exhaust enriched reduction media consisting of HC, CO and H2.

DeSOx Mode

A further regeneration mode is provided by the sulphur removal of the NOx storage catalytic converter (DeSOx Mode). This is necessary as the sulphur contained in the fuel causes sulfate formation which slowly deactivates the NOX storage catalytic converter.

The de-sulphurization procedure is designed for a sulfur content of 15 ppm parts per million (ppm)

Due to the high thermal stability of the sulfates, significant levels of sulphur reduction are only possible at temperatures above 620 C (1150 F).

The sulphur reduction procedure has been designed so that the storage capacity of the catalytic converter can mostly be restored without irreversible damage to the storage material.

The sub-stoichiometric mode is very demanding in terms of engine management. To be able to set air mass and exhaust gas recirculation independently on each other, two separate control circuits are used. The air mass is set using the intake manifold throttle valve. The exhaust recirculation rate is set using a new, model-based regulation concept.



A suitable combination of high pressure and low pressure EGR, with corresponding compression temperatures, enable stable rich operation even in the low load range with the fuel qualities that are typical for the USA. In addition to this, the injection strategy for the rich mode is changed. Up to six injections are used depending on characteristic values to attain a stable and low-soot combustion. This is particularly important in the sulphur reduction process to prevent soot accumulation in the particulate filter.



To attain the necessary exhaust gas temperatures in DeSOx operation, the conflict of interests between the component protection of the turbocharger and the higher sulfur-reduction performance was resolved using very late, non-combustion post-injection. The fuel partially reacts at the oxidation catalytic converter with the residual oxygen contained in the exhaust gas and therefore creates residual heat for the sulfur reduction of the NOx storage catalytic converter. These interventions in engine management are regulated to a neutral torque, meaning that the process has no noticeable effect on driving characteristics. As shown in the figure below, the regeneration intervals depend on the corresponding load conditions of the NOx storage catalytic converter with sulfur, nitrogen oxide or the soot load of the particulate filter. The maximum load conditions were adjusted to the permissible operating thresholds of the components.



DeNOx Concept

Taking the necessary engine operation and regeneration conditions as well as the catalytic converter properties into consideration, the corresponding regeneration mode is prioritized by a coordination program in the engine control module.

DeNOx regeneration is given a higher priority than other regenerations to prevent thermal NOx desorption.

A loading and discharging model is stored in the engine control module for DeNOx regeneration. This maps the characteristics of the DeNOx storage catalytic converter. The load condition of the catalytic converter is modeled during engine operation that is dependent on the exhaust temperature and volume velocity as well as the calculated raw NOx emissions. If the NOx load value exceeds a threshold value which represents the optimum conversion rate for the catalytic converter, the regeneration is conducted when the operating condition of the engine permits a regeneration mode to be activated.

Two criteria, which relate to the lambda signal or a NOx discharge model, are available for determining the end of regeneration.

As soon as the lambda sensor detects a rise in the reduction medium after the NOx storage catalytic converter, it is free of nitrogen oxide and regeneration has ended.

Due to cross-sensitivity of the lambda probe, this criteria is not permissible under a certain threshold temperature. For this reason, the discharge of the NOx storage catalytic converter is also modeled on the basis of the requirement and provision of reduction medium to reduce the stored NOx.

Sulfur reduction concept

The requirement for a DeSOx mode is necessitated by the sulfur load of the NOx storage catalytic converter and is calculated from fuel consumption and the sulfur content of the fuel.

To shorten the heating cycle of the exhaust system, sulfur reduction in the NOx storage catalytic converter is only conducted at the end of a particulate filter regeneration cycle. After reaching the desulphurisation temperature, the engine begins to use a long time-limited rich phase to enable an efficient desulphurization. The rich mode will be periodically interrupted to prevent excessive soot accumulation in the particulate filter. It is also interrupted when reaching a high exhaust gas temerature threshold. Likewise, this process will be interrupted at very low and very high engine loads.

The sulfur discharge is calculated in the engine control module. It depends on the sulfur load, the lambda value and the exhaust gas temperature. The de-sulfurization process will be ended by reaching the lower sulfur load threshold of the maximum time period.

Short lean-rich pulses ("wobbling")

Long time-limited rich phases ("LZF")



The H_2S catalytic converter, which was specially developed for this application, is placed downstream of the NOx storage catalytic converter and converts the H_2S , which is created during the DeSOx regeneration mode, completely into SO2.

The duration of the sulfur reduction process depends on the speed of sulfur reduction that is calculated for the NOx storage catalytic converter. This, in turn, depends on the lambda ratio and the temperature as it is calculated by the engine control module. The sulfur discharge is calculated in the engine control module. It depends on the sulfur load, the lambda value and the exhaust temperature.

The de-sulfurization process will be ended by reaching the sulfur load threshold or the maximum duration.



Preheating System

The 2.0 Liter TDI engine with common rail injection system has a diesel quick-start preheating system. This system allows an immediate "spark-ignition" start without long preheating time in virtually any climactic condition.

Advantages of the preheating system:

- "Spark engine" start at temperatures to -11.2°F (-24°C).
- Extremely quick preheating time. Within two seconds a temperature of up to 1832°F (1000°C) is reached on the glow-plug.
- Controllable temperatures for preheating and postheating.
- Self-diagnostic capability.
- Part of the On-Board Diagnosis Preheating System.



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Function

Preheating

The steel glow plugs are activated by the Engine Control Module (ECM) J623 over the Automatic Glow Time Control Module J179 in phase displacement with the aid of a pulse-width modulated (PWM) signal. The voltage on the individual glow plugs is adjusted over the frequency of the PWM impulses. For quick start with an ambient temperature of less than 64°F (18°C), a maximum voltage of 11.5 volts is present during preheating. This ensures that the glow plug heats up as quickly as possible (maximum two seconds) to over 1832°F (1000°C), thus reducing the preheating time of the engine.

Post-Heating

The PWM signal is reduced to 4.4 volts for post heating.

Post-heating is performed up to a coolant temperature of 64°F (18°C) after the engine start for a maximum of five minutes. Post-heating helps reduce hydrocarbon emissions and combustion noise during the engine warm-up phase.

Phase-Displaced Activation of the Glow Plugs

To relieve the vehicle electrical system voltage during the preheating phases, the glow plugs are activated in phase displacement. The falling signal flank always controls the next glow plug.



Special Tools

Designation	ΤοοΙ	Use
T10172/9 Adapter	5403 113	Adapter for work piece holder T10172
T10377 Assembly Sleeve	3403_113	For assembly of the O-ring on
	5403 068	the injection nozzle
T10384 Ratchet Ring Wrench	5403 114	For removal and installation of the diesel particulate filter

Designation	ΤοοΙ	Use
T10385 Insert Tool		For removal and installation of the exhaust gas return pipe
T40064/1 Pressure Piece	5403_112	Pressure piece for extractor
140004/111essure nece		T40064 for removal of the toothed-belt wheel for the high- pressure pump
T40094 Camshaft Insert Tool		For removal and installation of
T40094/1 Fixture	/2 /11	the camshaft
T40094/2 Fixture		
T40094/9 Fixture	A ROL	
T40094/10 Fixture		
T40094/11 Cover	/9 /1 /1	
	/10	
	\$403_063	

Designation	Tool	Use
T40095 Clamp		For removal and installation of the camshaft
	S403_064	
T40096/1 Chuck		For securing the divided camshaft wheel during installation and removal of the camshaft
T40159 Insert Tool with Ball	5403_000	For assembly work on the intake
Head		manifold

Designation	Tool	Use
T10401 Socket		For removal and installation of the EGR Cooler temperature sensor

An on-line Knowledge Assessment (exam) is available for this Self-Study Program.

The Knowledge Assessment may or may not be required for Certification.

You can find this Knowledge Assessment at:

www.vwwebsource.com

For Assistance, please call:

Volkswagen Academy

Certification Program Headquarters

1-877-491-4838

(8:00 a.m. to 8:00 p.m. EST)

Or, E-Mail:

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Volkswagen of America, Inc. 3800 Hamlin Road Auburn Hills, MI 48326 Printed in the U.S.A. April 2008

